

Mohamed Hadj Meliani<sup>1,2</sup>, Zitouni Azari<sup>2</sup>, Guy Pluvinage<sup>2</sup>, Yuri G. Matvienko<sup>3</sup>

## TWO-PARAMETER FRACTURE CRITERION BY VOLUMETRIC METHOD APPROACH DVOPARAMETARSKI KRITERIJUM LOMA PRISTUPOM VOLUMETRIJSKOG METODA

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Author's address:

<sup>1</sup>) LPTPM, FSSI, Hassiba Ben Bouali University of Chlef, Esalem City, Chlef, Algeria, [hadjmeliani@univ-metz.fr](mailto:hadjmeliani@univ-metz.fr)

<sup>2</sup>) Laboratoire de Mécanique, Biomécanique, Polymères, Structures, LaBPS-ENIM, Université Paul Verlaine de Metz, France

<sup>3</sup>) Laboratory of Modelling Damage and Fracture, Mechanical Engineering Research Institute of the Russian Academy of Sciences, Moscow, Russia

### Keywords

- notch stress intensity factor
- effective distance
- master curve
- volumetric method

### Abstract

A two-parameter fracture criterion has been proposed to predict fracture conditions of notched components. This criterion includes the critical notch stress intensity factor  $K_{\rho,c}$  which represents fracture toughness of a material with a notch of radius  $\rho$ , and the non-vanishing parameter from the Williams equation,  $A_3$  term. The effective  $A_3$ -term  $A_{3eff}$  has been estimated as the average value of the  $A_3$  term distribution in the region ahead of the notch tip at the effective distance  $X_{ef}$ . These parameters are derived from the volumetric method of notch fracture mechanics. The material failure curve or master curve  $K_{\rho,c} = f(A_{3eff,c})$  is established as a result of notched specimen tests. It is shown that the notch fracture toughness is a linear decreasing function of the  $A_{3eff,c}$ -stress. The use of the material failure curve to predict fracture conditions is demonstrated on gas pipes with the longitudinal surface notch.

### INTRODUCTION

For brittle mode I fracture in mainly elastic regime, current fracture assessment methodology relies on plane strain fracture toughness,  $K_{IC}$ , which is assumed to be a material property [1-4]. Recent studies [5-10] have shown that fracture toughness can be strongly affected by specimen size, crack depth and loading configuration. This dependency is often referred to the effect of crack-tip constraint. However, the fracture toughness estimated from standard test specimens may lead to unduly conservative results. It is commonly accepted that standard specimens in

### Ključne reči

- faktor intenziteta napona zarezata
- efektivno rastojanje
- master kriva
- volumetrijska metoda

### Izvod

Predložen je dvoparametarski kriterijum loma za procenu uslova loma kod komponenata sa zarezom. Ovaj kriterijum sadrži kritični faktor intenziteta napona zarezata  $K_{\rho,c}$  koji predstavlja žilavost loma materijala sa zarezom radijusa  $\rho$ , i ne iščezavajući parametar iz jednačine Vilijemsa, član  $A_3$ . Efektivni član  $A_3$ ,  $A_{3eff}$  je procenjen kao prosečna vrednost raspodele člana  $A_3$  u oblasti ispred vrha zarezata, na efektivnom rastojanju  $X_{ef}$ . Ovi parametri su izvedeni volumetrijskom metodom iz mehanike loma zarezata. Kriva otkaza materijala ili master kriva  $K_{\rho,c} = f(A_{3eff,c})$  je određena iz rezultata ispitivanja zarezanih epruveta. Pokazuje se da je žilavost loma zarezata linearno opadajuća funkcija napona  $A_{3eff,c}$ . Upotreba krive otkaza materijala radi procene uslova loma data je na primeru gasnog cevovoda sa podužnim površinskim zarezom.

laboratory testing are typically of high constraint while non-standard specimens and actual cracked structures may be low constraint configurations. The ASTM E-399 [11] testing procedure recommends certain types of specimen geometries and  $K_{IC}$  can be considered as the plane-strain fracture toughness. All specimen geometries recommended by ASTM E-399 are high constraint. Using the recommendation specimen geometry for testing creates an "ASTM Window" since their corresponding  $T$  or  $A_3$  values are within a certain range. The  $A_3$  quantifies the third term of

Williams stress field expansion /5/. A  $K_{IC}$  value is believed to represent a lower limiting value of fracture toughness and the ASTM E-399 may not be generally valid. Increasing the size of a specimen shifts the stress distribution closer to the K-stress. Consequently, larger specimens tend to possess better K-dominance. This may explain why a large specimen is better suited for ASTM fracture toughness  $K_{IC}$  testing in addition to the reason for the plastic zone size. This phenomenon limiting the recommendation of ASTM and can be explained using the analytical  $K-T$  or  $K-A_3$  relation for common effects of specimen geometries.

For the discussion above, it is now well-known in fracture mechanics community that the single fracture parameter alone may not be adequate to describe the crack-tip condition. To address this problem, there has been a recent surge of interest in crack-growth behaviour under conditions of low crack-tip stress triaxiality. This paper exploited the  $K-A_3$  crack approach which is derived from a rigorous asymptotic solution and is developed for a two-parameter fracture. With  $K$  as the driving force and  $A_3$  a constraint parameter, this approach has been successfully used to quantify the constraints of notch-tip fields for various proposed geometry and loading configurations.

#### EXTEND OF SINGLE PARAMETER FRACTURE MECHANICS (SPFM)

##### *Classical Theoretical Background and motivation*

Many researchers have long advocated more pragmatic, engineering approach to assess the fracture integrity of cracked structures /13/. This approach requires that constraint in the test specimen approximate that of the structure to provide an "effective" toughness for use in a structural integrity assessment. The appropriate constraint is achieved by matching thickness and crack depth between specimen and structure.

##### *The $K-A_3$ crack approach*

A detailed stress analysis is carried out in the vicinity of the crack front, in order to emphasize the characteristics of the two dimensional stress field. Due to that fact that in mode I, the crack propagation is determined by the stress perpendicular to the crack, namely  $\sigma_{yy}$ , this stress component is dealt with in the analysis. The linear elastic Williams' series solution is used for describing the crack tip stress fields. Three terms of Williams' solution, quantified by two parameters,  $K$  for the intensity of the stress field,  $T$  and  $A_3$  for the crack tip constraint, are studied and found to be sufficient for representing the crack tip stress distribution. Similar three term approach is used for the interpretation of brittle fracture under quasi-static conditions (Chao and Zhang /6/; Chao and Reuter /47/, Chao et al. /7, 8/; Liu and Chao /48/). Note that the model de RKR – model describing the micromechanism of brittle fracture due to Ritchie, Knott and Rice /49/ proposes that crack growth in brittle fracture takes place when tensile stress at a critical distance  $r_c$  ahead of the notch-tip reaches a critical value  $\sigma_c$ . The conventional relation for mode I fracture

$$K_I = K_{IC} \quad (1)$$

can be considered as a simple representation of the RKR model. In Eq. (1), the mode I stress intensity factor  $K_I$  describes the state of stress near the crack-tip and fracture toughness  $K_{IC}$  is a material parameter representing the critical conditions required for fracture initiation. Numerous authors consider that the critical stress failure as a very promising and convenient fracture criterion, especially for running cracks or cracks in pipes. The maximum tangential stress, is often assumed, takes place along the line of the initial crack. Since the T-stress vanishes in the tangential stress along this direction, its effect on initiation of brittle fracture is normally ignored. Ayatollahi et al. /50/ reveal that the maximum along  $\sigma_{\theta\theta}$  is not always zero $_{\theta=0}$  and angular deviation can occur only for positive values of T-stress. When the T-stress is negative the maximum  $\sigma_{\theta\theta}$  is always along in the direction of propagation,  $\theta=0$ . In mode I, fracture occurs when the tangential stress  $\sigma_{\theta\theta}$  at some point along  $\theta_{max}$  at a critical distance  $r_c$  from the crack-tip, exceeds the critical or maximum stress  $\sigma_c$ . The tangential stress  $\sigma_{\theta\theta}$  near the tip is rewritten as

$$\begin{aligned} \sqrt{2\pi r} \cdot \sigma_{\theta\theta} = & K_I \cos^3 \frac{\theta}{2} + T \sqrt{2\pi r} \cdot \sin^2 \theta + \\ & + \frac{3}{4} A_3 \sqrt{2\pi r} \cdot \left( 5 \cos \frac{\theta}{2} - \cos \frac{5\theta}{2} \right) \end{aligned} \quad (2)$$

we can note  $K_{app} = \sigma_{\theta\theta} \sqrt{2\pi r}$ , with  $K_{app}$  representing the apparent stress intensity factor. At fracture, critical apparent stress intensity factor  $K_{app,c}$ , using Eq. (2), one then has

$$\begin{aligned} K_{app,c} = (\sigma_{\theta\theta})_{max} \sqrt{2\pi r_c} = & K_c \cos^3 \frac{\theta_c}{2} + T_c \sqrt{2\pi r_c} \cdot \sin^2 \theta_c + \\ & + \frac{3}{4} A_{3c} \sqrt{2\pi r_c} \cdot \left( 5 \cos \frac{\theta_c}{2} - \cos \frac{5\theta_c}{2} \right) \end{aligned} \quad (3)$$

For mode I, assuming three terms are sufficient to characterise the crack-tip stress field; we examine the case when crack dose not curve, i.e. the second term in Eq. (2) or the T-stress vanishes. Equation (3) becomes

$$K_{app} = \sigma_{\theta\theta, \theta=0} \sqrt{2\pi r} = K_I + 3\sqrt{2\pi} \cdot A_3 \cdot r \quad (4)$$

The  $A_3$  term in the Williams series expansion is determined by employing finite element analysis and Eq. (4) can be rewritten for high order as

$$K_{app} = \sigma_{\theta\theta, \theta=0} \sqrt{2\pi r} = K_I + B_3 r + B_5 r^2 \quad (5)$$

where  $B_3 = 3\sqrt{2\pi} A_3$  and  $B_5 = 5\sqrt{2\pi} A_5$ . If the stress intensity factor distribution expressed by the left side of Eq. (5) is plotted against the distance from the crack-tip  $r$ , a linear fit to the data will yield the slope  $m$ . Then,  $A_3$  can be obtained from the slope  $m$ , that is  $A_3 = m/3\sqrt{2\pi}$ . For the distribution of the stress intensity factor in Eq. (5), the fit is a parabolic curve. Having both  $K_{app}$  and  $A_3$  determined the three term solutions in Eq. (4) give the near crack-tip stress.

##### *Volumetric method mesofracture to extend SPFM to notch*

Although many works have carried out estimation for the stress intensity factor with the presence of T-stress of pipeline, they have exclusively focused on classical fracture

mechanics with crack to estimate the toughness. We present notch fracture mechanics (NFM) principles applied to study stress distribution at the notch tip of pipes submitted to internal pressure. Volumetric Method, presented by Pluvinage /51/ is a meso-mechanical method belonging to this NFM. It is assumed, according to the mesofracture principle that the fracture process requires a physical volume. This assumption is supported by the fact that fracture resistance is affected by loading mode, structural geometry, and scale effect. By using the value of the “hot spot stress” i.e. the maximum stress value, it is not possible to explain the influence of these parameters on fracture resistance. It is necessary to take into account the stress value and the stress gradient in all neighbouring points within the fracture process volume. This volume is assumed to be quasi-cylindrical with a notch plastic zone of similar shape. The diameter of this cylinder is called the “effective distance”. By computing the average value of stress within this zone, the fracture stress can be estimated, this leads to a local fracture stress criterion based on two parameters, the effective distance  $X_{eff}$  and the effective stress  $\sigma_{eff}$ , the graphical representation of this local fracture stress criterion is given in Fig. 1, where the stress normal to the notch plane is plotted against the distance ahead of notch. For determination of  $X_{eff}$ , a graphical procedure is used; it has been observed that the effective distance is related to the maximum value of the relative stress gradient  $\chi$ . This distance corresponds to the beginning of the pseudo stress gradient is indicated in Fig. 2. The opening stress distribution at the notch is calculated using FEM for elastic analysis of 2D model in plane strain conditions, the effective distance  $X_{eff}$  is determined using normal stress distributions below. The notch root, plotted in bi-logarithmic axes; the relative stress gradient (see Eq. (6)), plotted on the same graph, allows obtaining an effective distance precise value.

$$\chi(r) = \frac{1}{\sigma_{yy}(r)} \frac{\partial \sigma_{yy}(r)}{\partial r} \quad (6)$$

where  $\chi(r)$  and  $\sigma_{yy}(r)$  are the relative stress gradient and maximum principal stress or crack opening stress, respectively. The relative stress gradient depicts the severity of the stress concentration around the notch and crack tips. However, the stress distribution effect is not solely a major parameter for the fracture process zone. The minimum point of the relative stress gradient in the bi-logarithmic diagram is conventionally taken into account as the relevant effective distance and signifies the virtual crack length. The effective stress is defined as the average of the weighted stress inside the fracture process zone:

$$\sigma_{eff} = \frac{1}{X_{eff}} \int_0^{X_{eff}} \sigma_{yy}(r) \Phi(r) dr \quad (7)$$

where  $\sigma_{eff}$ ,  $X_{eff}$ ,  $\sigma_{yy}(r)$  and  $\Phi(r)$  are effective stress, effective distance, maximum principal stress and weight function, respectively. The unit weight function and Peterson's weight function are the simplest definitions of weight function the effective distance. The unit weight function deals with the average stress and Peterson's weight function gives the stress value at a specific distance and it is not required to compute

numerical integration. Therefore, the noted Notch Stress Intensity Factor, NSIF, is described and is defined as a function of effective distance and effective stress given by relationship

$$K_\rho = \sigma_{eff} \sqrt{2\pi X_{eff}} \quad (8)$$

the NSIF is considered as a value of fracture toughness with units  $\text{MPa}\cdot\text{m}^{0.5}$ , and the minimum effective distance corresponds to the abscissa of the upper limit of zone II and its distance from notch root is suggested to be the effective distance  $X_{eff}$ . The effective T-stress,  $T_{eff}$  is not singular as  $r \rightarrow 0$ , but it can modify to the effective crack tip plastic zone.  $T_{eff}$  can be rewritten as

$$T_{eff} = \frac{1}{X_{eff}} \int_0^{X_{eff}} T_{xx}(r) \Phi(r) dr \quad (9)$$

with  $T_{xx} = (\sigma_{xx} - \sigma_{yy})$  and the effective stress intensity factors take the form of

$$K_{eff} = K_\rho + T_{eff} \sqrt{2\pi X_{eff}} = \sigma_{eff} \sqrt{2\pi X_{eff}} + T_{eff} \sqrt{2\pi X_{eff}} \quad (10)$$

as  $r \rightarrow X_{eff}$

and the stress biaxiality ratio in the effective distance can be rewritten as

$$\beta_{eff} = \frac{T_{eff} \sqrt{2\pi X_{eff}}}{K_\rho} \quad (11)$$

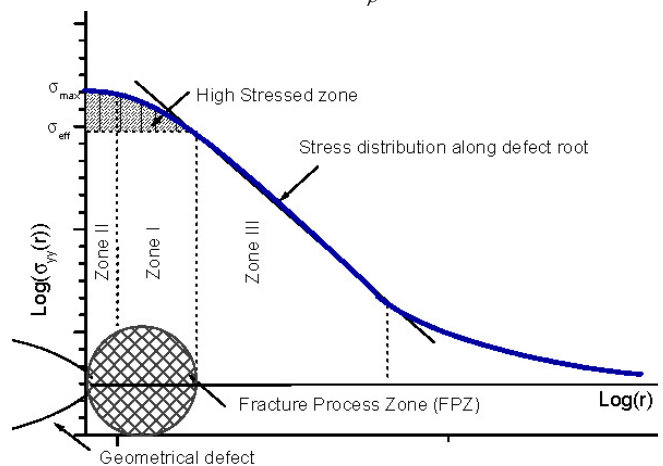


Figure 1. Schematic presentation of a local stress criterion for fracture emanating from notches.

Slika 1. Shema kriterijuma lokalnog napona za lom koji potiče od zareza

The Notch Stress Intensity Factor  $K_\rho$ , the effective T-stress,  $T_{eff}$ , near the notch root and the relative distance are shown in Fig. 2. Effective T-stress is used as constraint parameter. This addition to the classical plastic notch tip parameter  $K_\rho$  provides an effective two-parameter characterization of elastic notch-tip fields in a variety of notch configurations and loading conditions. With the volumetric method, we examine the case when crack does not curve, i.e. the second term in Eq. (3) or the T-stress vanishes. Equation (4) becomes

$$K_{eff}^c = \sigma_{\theta\theta, \theta=0} \sqrt{2\pi X_{eff}} = K_\rho + 3\sqrt{2\pi} \cdot A_{3eff} \cdot r \quad (12)$$

The  $A_{3eff}$  term in the Williams' series expansion is determined by finite element analysis.

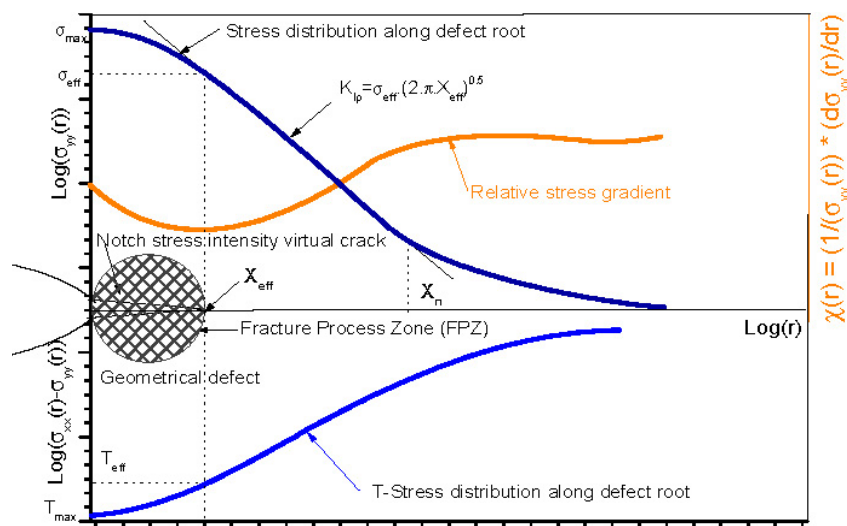


Figure 2. Determination of effective stress intensity factor and the effective T-stress at notch root together with the relative stress gradient versus distance from the notch tip.

Slika 2. Određivanje efektivnog faktora intenziteta napona i efektivnog T-napona u korenu zareza zajedno sa relativnim gradijentom napona u zavisnosti od rastojanja od vrha zareza

### Finite Element Modelling

The geometry considered in this study is a cylinder with a V-shaped longitudinal surface notch subject to different internal pressure  $P$ , as shown in Fig. 3. The effect of three parameters: ratio of inner radius of the cylinder to the thickness,  $R_i/t$ , the ratio of the notch depth to cylinder thickness,  $a/t$ , and pressure on T-stress and Stress Intensity factors (SIF) is systematically considered. To cover practical and interesting ranges of these three variables, four different values of  $R_i/t$ , 5, 10, 20 and 40, are selected. In terms of crack depth, four different values of  $a/t$  are selected, ranging from  $a/t = 0.1$  to 0.75. In terms of pressure, four different values of  $p$  are selected, ranging from pressure of 20 to 50 bar. Thus, a total of 84 different experimental setups are considered in this investigation, the details of which are listed in Table 1.

The finite element method is used to determine notch-tip parameters  $T$  and  $SIF$  for the pipe specimens. The structure are modelled by CASTEM 2000 code in two dimensions under plane strain conditions using free meshed isoperimetric triangular elements only on half of the specimen. The elastic analyses comprise 31485 elements and 63526 nodes. A fanlike mesh focused at the notch-tip is employed, because this yields more accurate non-singular terms estimates. Further, a more detailed mesh sensitivity study has shown that further refinement of the mesh leads to only small changes ( $< 1\%$ ), the pipe specimen geometry is illustrated in Fig. 3. The wall thickness is 10 mm, the length of the pipe is 40 mm, dimensions of the notch are presented in Fig. 4a. Support and symmetric boundary conditions are used in this model.

Table 1. List of analysis cases for the FE analysis.

Tabela 1. Skup slučajeva za FE analizu

$P$	$R_i/t$	$a/t$	$P$	$R_i/t$	$a/t$	$P$	$R_i/t$	$a/t$	$P$	$R_i/t$	$a/t$
20	5	0.1	20	10	0.1	20	20	0.1	20	40	0.1
		0.3			0.3			0.3			
		0.5			0.5			0.5			
		0.75			0.75			0.75			
30	5	0.1	30	10	0.1	30	20	0.1	30	40	0.1
		0.3			0.3			0.3			
		0.5			0.5			0.5			
		0.75			0.75			0.75			
40	5	0.1	40	10	0.1	40	20	0.1	40	40	0.1
		0.3			0.3			0.3			
		0.5			0.5			0.5			
		0.75			0.75			0.75			
50	5	0.1	50	10	0.1	50	20	0.1	50	40	0.1
		0.3			0.3			0.3			
		0.5			0.5			0.5			
		0.75			0.75			0.75			

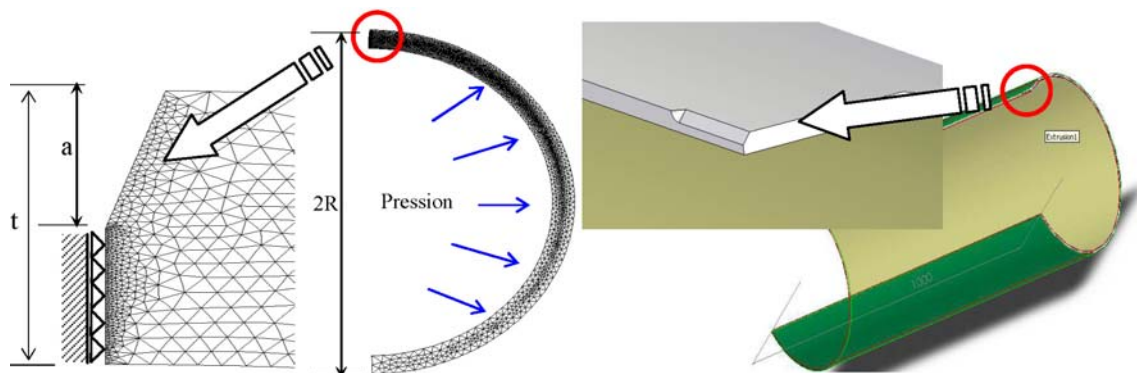


Figure 3. (a) Geometry boundary conditions and loading configuration using the half of pipe. (b) Typical 2D finite element mesh used to model the crack on the half pipeline for elastic analysis.

Slika 3. (a) Geometrija graničnih uslova i konfiguracija opterećenja za jednu polovinu cevi. (b) Tipična 2D mreža konačnih elemenata za modeliranje prsline na polovini cevovoda za elastičnu analizu

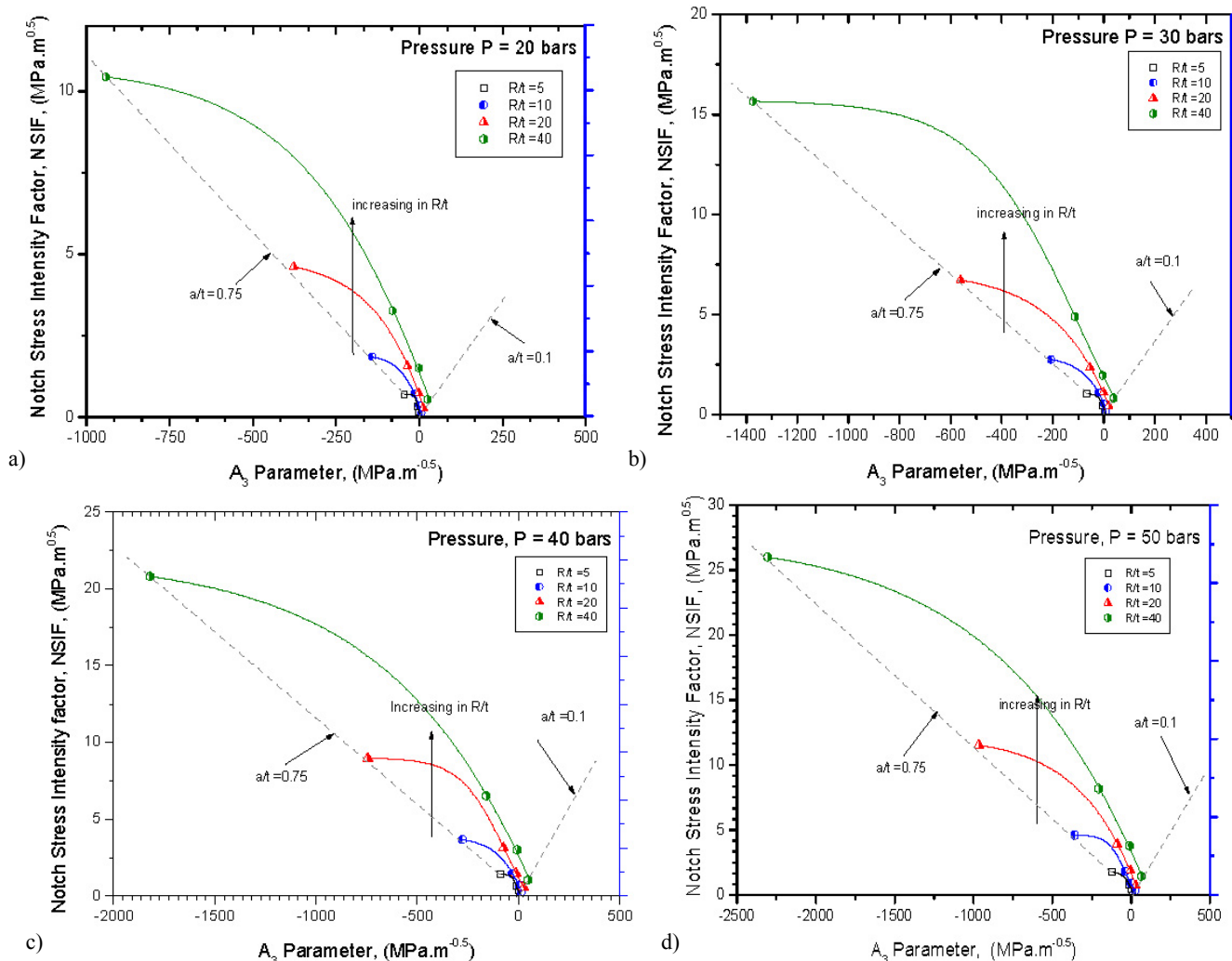


Figure 4. Development of notch tip constraint represented by  $A_3$  for various situations of pressure and diameters of Pipeline.  
 Slika 4. Razvoj veze na vrhu zarezta predstavljeno sa  $A_3$  za razne slučajeve pritiska i prečnika cevovoda

INTERPRETATION OF RESULTS AND DISCUSSION

A detailed stress analysis is carried out in the vicinity of the notch front, in order to emphasize the characteristics of the two-dimensional stress field. Due to the fact that in Mode I, crack propagation is determined by the stress perpendicular to the notch, namely  $\sigma_{yy}$ , this stress compo-

nent is dealt within the analysis. The coefficients of the higher order stress terms represent one part of a larger database which shall also include information on various constraint parameters. Since this study is concentrated on elastic two-dimensional notches and initial part of the finite element constraint parameter are investigated.



Recently, some publications have carried out a complete analysis of higher order crack fields in power-law hardening materials and have shown that a two-term expansion is not sufficient to describe the near tip fields while more than three terms are redundant. In this application, the effects of pipeline size, loading configuration and notch depth are presented.

In order to investigate the effect of pressure on the constraint for notch-tip field, the developments of Notch Stress Intensity Factors at different pressure ( $P = 20, 30, 40$  and  $50$  bar) are modelled and the results are shown in Fig. 4a-d. The data on  $K_p$  versus  $A_3$  are plotted on different diameters ( $R/t = 5, 10, 20$  and  $40$ ). The first remarks that  $K_p$  increase with increasing  $-A_3$ . For small notch length,  $a/t < 0.3$ , the values of  $A_3$  are positive for all pressure and pipe diameters. Since a pipe has large  $a/t$  ratios, this parameter takes very negative values. It can be seen that  $A_3$  has a linear function  $a/t$ , and increases with increasing the depth of the notch and pressure (see Fig. 5 for example). This observation is in contradiction with the remarks of Lam et al. [52]. The authors [52] justified the constant value of  $A_3$  when the pressure increases by the Large Scale Yielding (LSY) condition.

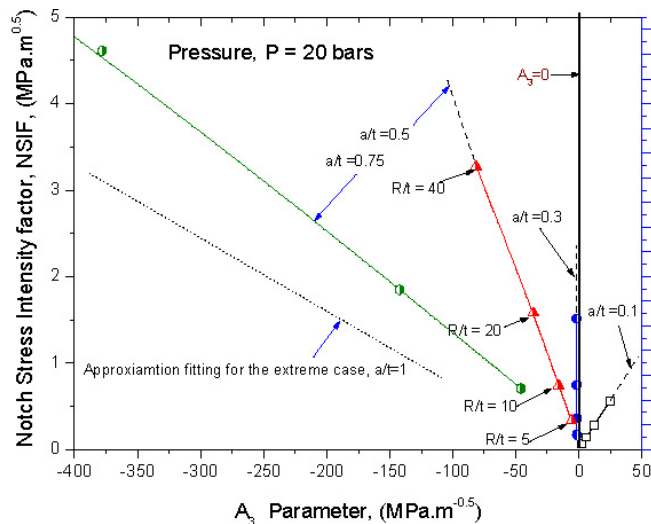


Figure 5. Example of linear relationship between the  $A_3$  parameter and the pipe diameter for different  $a/t$  ( $P = 20$  bar).

Slika 5. Primer linerane veze parametra  $A_3$  i prečnika cevovoda za različite  $a/t$  ( $P = 20$  bar)

Figure 6 shows the situation of the higher order terms near the crack-tip. In this work we only consider the opening stress along the zero degree direction ahead of the notch tip. That is, we assumed that the crack would propagate along the zero degree direction. For any flawed specimen or structure, a notch driving force may be established by running a linear elastic FEA for the geometry and any applied load to determine the pair  $(K_p, A_3)$ . For a surface notch, the crack front is a curved line. A  $(K_p, A_3)$  pair and a notch driving force at each point along the notch front are present as the far field load is increased. By putting the entire notch driving forces along the notch front together, the notch driving force for the surface notch becomes a curved front as depicted in Fig. 7.

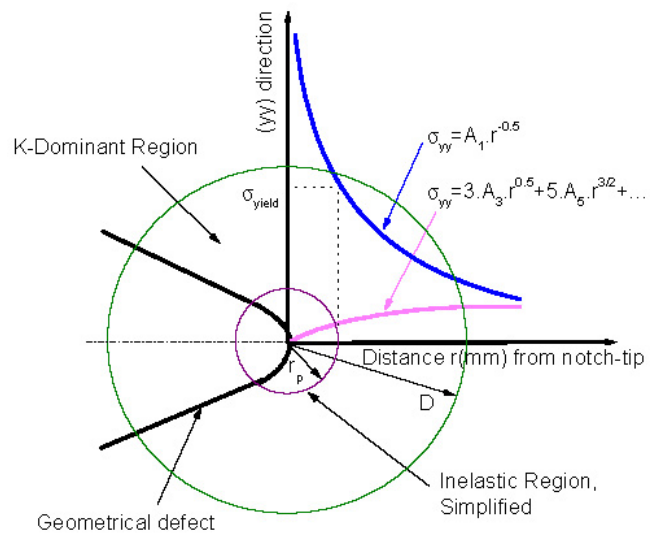


Figure 6. The concept of the extended SFEM.

Slika 6. Konceptija proširene SFEM

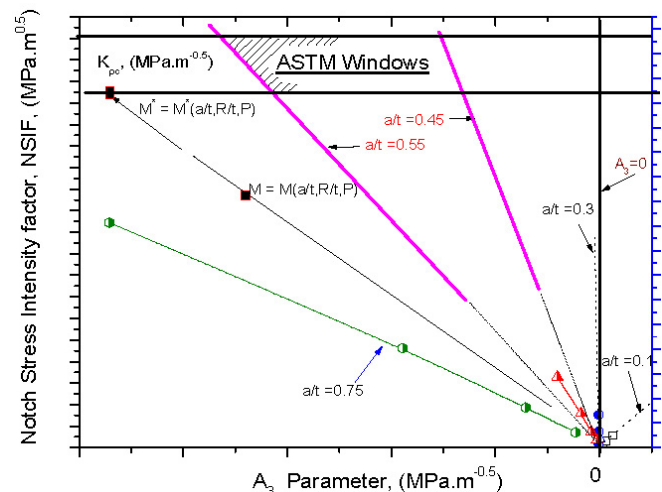


Figure 7. Schematic presentation of  $A_3$  variation versus NSIF.

Slika 7. Shematski prikaz promene  $A_3$  sa NSIF

Since  $(K_p, A_3)$  varies from point to point along the notch front for a given far field load. Each point on the curve represents the condition of the opening stress at a particular point in the notch front of the surface crack. For instance, the left point of the curve may represent the condition of the opening stress at the deepest point of the surface notch and the right point corresponding to the point where the crack front intersects with the surface.

For common specimen geometries, the relation between  $K$  and  $A_3$  is tabulated from numerical calculations for convenience. Since the ratio between  $K$  and  $A_3$  is a constant for a given geometry. In Fig. 7, point  $(0,0)$  represents the condition of no applied load. The two points  $(0,0)$  and  $(K-A_3)$  from the driving force are represented in a straight line in the  $K-A_3$  plane and the intersection of the notch driving force with the material failure line yields the predicted  $K_C$  and  $A_{3C}$  for the particular flawed structure.

## CONCLUSIONS, REMARKS

The  $K-A_3$  methodology is used and  $A_3$  parameter is identified to quantify the constraint at the notch tip. Procedures to shift the mechanical properties curve between Pipelines of different in-plane constraint levels are developed which enables the determination of the transition curve of non-standard flawed structures from the experimental results of standard specimens. The magnitudes of negative  $A_3$  parameters are greater than those of large diameters of pipelines. The low constraint due to relatively large magnitude of negative  $A_3$  term may be expected to inhibit the crack extension in the same plane and promote crack kinking. Further, among the small diameters, the less  $A_3$  exhibit lower apparent  $K$ .

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